CLEAN VERSION

IN THE DRAWINGS:

Kindly see the Drawing Change Authorization Request filed concurrently herewith.

IN THE SPECIFICATION:

Please amend the specification pursuant to 37 C.F.R. 1.121 as follows (see the accompanying "marked up" version pursuant to 1.121):

On Page 3, lines 1-5, please amend as follows:

During operation, a drive force of motor 105 causes flywheel 111 to rotate. Drive shaft 112 rotates when the clutch in the clutch/break mechanism (not shown), mounted in flywheel 111, connects. Drive shaft 112 rotates main gear 114. Main gear 114 rotates crank shaft 113. Connecting rod 106 on eccentric portion 113a causes slide 104 to operate.

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BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1(A) is a front-view of a press according to the present invention.

Fig. 1(B) is a side-view of a press according to the present invention.

Fig. 2 is a partial cross-section-drawing-along line II-II in Fig. 1(A).

Fig. 3(A) is a detailed side-view of a drive mechanism according to the present

invention.

Fig. 3(B) is a cross-section drawing along the III-III line in Fig. 3(A).

Fig. 3(C) is a cross-sectional drawing along the III-III line at a bottom dead center

position.

Fig. 3(D) is a is a cross-sectional drawing along the III-III line at a top dead center

position.

Fig. 4(A) is a front-view of a conventional press.

Fig. 4(B) is a side-view of a conventional press.

5 is a partlat cross-section drawing along the I-I line in Fig. 4 (A):

On Page 10, line 20 to page 11, line 2, please amend as follows:

A guide 3 is integrally formed in frame 2. Guide 3 supports each side of slide 4.

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Guide 3 allows slide 4 to operate in a guided up-and-down manner. A bolster 8 is below slide 4. A bed 22 supports bolster 8 below slide 4. A die 7 is between slider 4 and bolster 8, and plate members 21, 21. In operation, a connecting rod 6 drives die 7 and slide 4, as will be explained.

On Page 11, lines 20-end, please amend as follows:

A slide-side gib 4a is on each side of slide 4 opposite press center P. Slide-side gibs 4a are supported on three sides (three mating surfaces (first, second, and third mating surfaces) as shown) by respective front liners 3a, side liners 3b, and rear liners 3c attached in respective guides 3. Front liners 3a, side liners 3b, and rear liners 3c accurately guide slide 4 along guide 3 during operation and maintain alignment with press center P and frame 2. Front liners 3a, side liners 3b, and rear liners 3c are symmetrically disposed, to the respective front, side, and rear of press center P.

On Page 12, lines 1-11, please amend as follows:

Cross sections 2a, 2b, 2c, and 2d of frame 2 are symmetrical to press center P. Cross sections 2a, 2b, 2c, and 2d are symmetrical to a first center line (vertical centerline of press 1) and symmetrical to a second center line (horizontal centerline of press 1), both shown on Fig. 2. During operation, press 1 may expand or contract due to operational and

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environmental pressures. Since cross sections 2a, 2b, 2c, and 2d are symmetrical to press center P any expansion during operation forced to be uniform along a front-back axis and left-right axis to press center P. It is to be understood, that controlling the effects of expansion, minimizes the possibility of operational errors and press 1 failure. It is to be understood that the effects of operational expansion are beneficially managed through a combination of frame integral construction, frame symmetry, alignment of press center P, slide 4 (with slide-side gibes 4a), and guide 3 in press 1 and other construction details indicated above and below.

On Page 13, line 20 to line 25, please amend as follows:

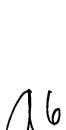
It should be understood that because the rotation axis of main gear 14 is eccentric to the rotation axis of crank shaft 13, in operation, slide 4 operates at a slow speed near the bottom dead center position, and at a higher speed in other positions. Thus, since the rotation axes of main gear 14 and crank shaft 13 are eccentric, through the operation of first link 15 and second link 17, this results in the different operation speeds of slide 4 through an operational cycle.

On page 14, kindly amend lines 11-15, as follows:

A center line exits (as shown) between the rotation center of crank shaft 13, main

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gear 14 of press 1. The difference between the center of crank shaft 13 and main gear 14 is a distance along the center line.

A rotation angle theta (θ) is formed (not shown) by main gear 14 during rotation relative to center line.

On page 14, kindly amend lines 18-21, as follows:

Additionally referring now to Figs. 3(C) and 3(D), an inner angle theta' (θ ') (a first angle) is defined between first link 15 and second link 17. Inner angle theta' (θ '), changes through the rotation of main gear 14 between an opening position at the bottom dead center position and a closing position at the top dead center position, as shown.

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